

Substituting the values in (i), we get

$$q = \frac{(1300 - 40)}{0.4843} = 2601.7 \text{ W/m}^2 \quad (\text{Ans.})$$

(ii) Maximum temperature to which common brick is subjected, t_2 :

$$q = h_i (t_{hf} - t_1) = \frac{(t_1 - t_2)}{L_A / k_A}$$

or, $2601.7 = 33.9 (1300 - t_1)$

or, $t_1 = 1300 - \frac{2601}{33.9} = 1223.27^\circ\text{C}$

and, $2601.7 = \frac{(1223.27 - t_2)}{(0.22/3.5)}$

or, $t_2 = 1223.27 - 2601.7 \times \frac{0.22}{3.5}$
 $= 1059.73^\circ\text{C} \quad (\text{Ans.})$

2.6. HEAT CONDUCTION THROUGH HOLLOW AND COMPOSITE CYLINDERS

2.6.1. HEAT CONDUCTION THROUGH A HOLLOW CYLINDER

Case I. Uniform conductivity :

Refer to Fig. 2.45. Consider a hollow cylinder made of material having constant thermal conductivity and insulated at both ends.

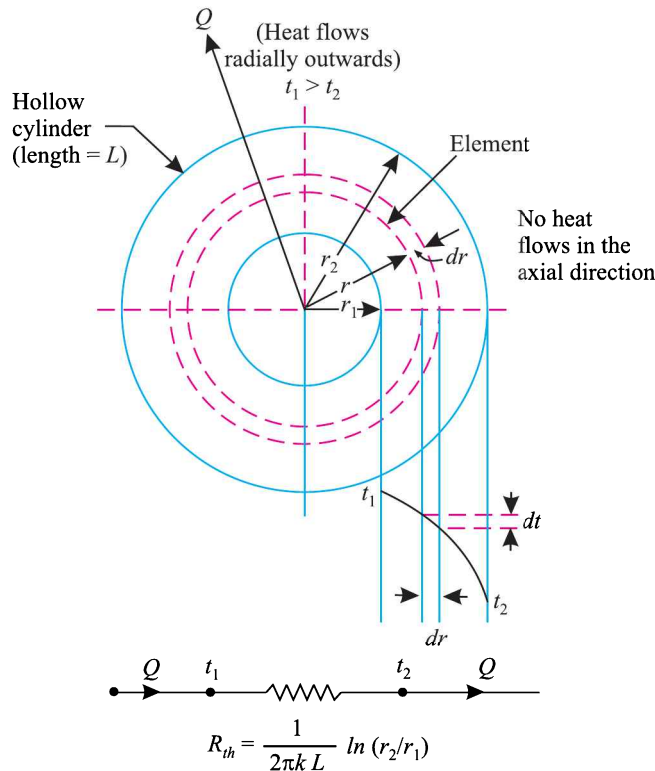


Fig. 2.45.

Let, r_1, r_2 = Inner and outer radii;

t_1, t_2 = Temperatures of inner and outer surfaces, and

k = Constant thermal conductivity within the given temperature range.

The general heat conduction equation in cylindrical coordinates is given by,

$$\left[\frac{\partial^2}{\partial r^2} + \frac{1}{r} \cdot \frac{\partial}{\partial r} + \frac{1}{r^2} \cdot \frac{\partial}{\partial \phi^2} + \frac{\partial^2}{\partial z^2} \right] + \frac{q_g}{k} = \frac{1}{\alpha} \cdot \frac{\partial t}{\partial \tau} \quad \dots(\text{Eqn. 2.22})$$

For steady state $\left(\frac{\partial t}{\partial \tau} = 0 \right)$, unidirectional [$t \neq f(\phi, x)$] heat flow in radial direction and with no internal heat generation ($q_g = 0$), the above equation reduces to

$$\frac{d^2 t}{dr^2} + \frac{1}{r} \cdot \frac{dt}{dr} = 0$$

or,
$$\frac{1}{r} \cdot \frac{d}{dr} \left[r \cdot \frac{dt}{dr} \right] = 0$$

Since, $\frac{1}{r} \neq 0$, therefore,
$$\frac{d}{dr} \left(r \cdot \frac{dt}{dr} \right) = 0$$

or,
$$r \cdot \frac{dt}{dr} = C \text{ (a constant)} \quad \dots(2.54)$$

Integrating the above equation, we get

$$t = C \ln(r) + C_1 \quad \dots(2.55)$$

(where C_1 = Constant of integration)

Using the following boundary conditions, we have :

At $r = r_1$, $t = t_1$; At $r = r_2$, $t = t_2$

\therefore
$$t_1 = C \ln(r_1) + C_1 \quad \dots(i)$$

$$t_2 = C \ln(r_2) + C_1 \quad \dots(ii)$$

From (i) and (ii), we have

$$C = -\frac{(t_1 - t_2)}{\ln(r_2 / r_1)} \text{ and } C_1 = t_1 + \frac{t_1 - t_2}{\ln(r_2 / r_1)} \ln(r_1) \quad \dots(2.56)$$

Substituting the values of these constants in eqn. (2.55), we have

$$t = t_1 + \frac{t_1 - t_2}{\ln(r_2 / r_1)} \ln(r_1) - \frac{(t_1 - t_2)}{\ln(r_2 / r_1)} \cdot \ln(r) \quad \dots(2.57)$$

[Equation 2.57 is the expression for *temperature distribution* in a hollow cylinder].

or,
$$(t - t_1) \ln(r_2 / r_1) = (t_1 - t_2) \ln(r_1) - (t_1 - t_2) \ln(r)$$

$$= (t_2 - t_1) \ln(r) - (t_2 - t_1) \ln(r_1) = (t_2 - t_1) \ln(r/r_1)$$

or,
$$\frac{t - t_1}{t_2 - t_1} = \frac{\ln(r/r_1)}{\ln(r_2/r_1)} \quad \text{(Dimensionless form)} \quad \dots(2.58)$$

From the above equation, the following points are worth noting :

- (i) The *temperature distribution is logarithmic* (not linear as in the case of plane wall).
- (ii) *Temperature at any point* in the cylinder can be expressed as a *function of radius only*. Isotherms (or lines of constant temperatures) are then concentric circles lying between the inner and outer boundaries of the hollow cylinder.
- (iii) The temperature profile [Eqn. (2.57)] is *nearly linear for values of (r_2/r_1) of the order of unity, but decidedly non-linear for large values of (r_2/r_1)*

Determination of conduction heat transfer rate (Q) :

The conduction heat transfer rate is determined by utilizing the temperature distribution [Eqn. (2.57)] in conjunction with Fourier's equation as follows :

$$\begin{aligned}
 Q &= -kA \frac{dt}{dr} \\
 &= -kA \frac{d}{dr} \left[t_1 + \frac{t_1 - t_2}{\ln(r_2/r_1)} \ln(r_1) - \frac{(t_1 - t_2)}{\ln(r_2/r_1)} \ln(r) \right] \\
 &\quad \text{[Substituting the value of } t \text{ from Eqn. (2.57)]} \\
 &= -k(2\pi r \cdot L) \left[\frac{-(t_1 - t_2)}{r \ln(r_2/r_1)} \right] \\
 &= 2\pi k L \frac{(t_1 - t_2)}{\ln(r_2/r_1)} = \frac{(t_1 - t_2)}{\frac{\ln(r_2/r_1)}{2\pi k L}} \left[= \frac{\Delta t}{R_{th}} \right] \quad \left(\text{where, } R_{th} = \frac{\ln(r_2/r_1)}{2\pi k L} \right)
 \end{aligned}$$

Hence,
$$Q = \frac{(t_1 - t_2)}{\frac{\ln(r_2/r_1)}{2\pi k L}} \quad \dots(2.59)$$

Alternative method :

Refer to Fig. 2.45 Consider an element at radius ‘r’ and thickness ‘dr’ for a length of the hollow cylinder through which heat is transmitted. Let dt be the temperature drop over the element.

Area through which heat is transmitted, $A = 2\pi r \cdot L$.

Path length = dr (over which the temperature falls is dt)

$$\begin{aligned}
 \therefore Q &= -kA \cdot \left(\frac{dt}{dr} \right) \\
 &= -k \cdot 2\pi r \cdot L \frac{dt}{dr} \text{ per unit time}
 \end{aligned}$$



Induction eddy current heating.

or,
$$Q \cdot \frac{dr}{r} = -k \cdot 2\pi L \cdot dt$$

Integrating both sides, we get

$$Q \int_{r_1}^{r_2} \frac{dr}{r} = -k \cdot 2\pi L \int_{t_1}^{t_2} dt$$

or,
$$Q [\ln(r)]_{r_1}^{r_2} = -k \cdot 2\pi L [t]_{t_1}^{t_2}$$

or,
$$Q \cdot \ln(r_2/r_1) = -k \cdot 2\pi L (t_2 - t_1) = k \cdot 2\pi L (t_1 - t_2)$$

∴
$$Q = \frac{k \cdot 2\pi L (t_1 - t_2)}{\ln(r_2 / r_1)} = \frac{(t_1 - t_2)}{\left[\frac{\ln(r_2 / r_1)}{2\pi k L} \right]} \quad \dots(2.60)$$

Case II. Variable thermal conductivity :

A. Temperature variation in terms of interface temperatures (t_1, t_2) :

The heat flux equation is given by

$$\begin{aligned} Q &= -kA \frac{dt}{dr} \quad [\text{where } k = k_0 (1 + \beta t)] \\ &= -k_0 (1 + \beta t) 2\pi \cdot r \cdot L \cdot \frac{dt}{dr} \end{aligned}$$

or,
$$Q \cdot \frac{dr}{r} = -k_0 \cdot 2\pi L (1 + \beta t) dt \quad \dots(2.61)$$

Integrating both sides, we have

$$Q \int_{r_1}^{r_2} \frac{dr}{r} = -k_0 \cdot 2\pi L \int_{t_1}^{t_2} (1 + \beta t) dt$$

$$Q [\ln(r)]_{r_1}^{r_2} = -k_0 \cdot 2\pi L \left[t + \beta \cdot \frac{t^2}{2} \right]_{t_1}^{t_2}$$

$$\begin{aligned} Q \cdot \ln(r_2 / r_1) &= -k_0 \cdot 2\pi L \left[(t_2 - t_1) + \frac{\beta}{2} (t_2^2 - t_1^2) \right] \\ &= -k_0 \cdot 2\pi L \left[(t_2 - t_1) + \frac{\beta}{2} (t_2 + t_1)(t_2 - t_1) \right] \\ &= k_0 \cdot 2\pi L \left[1 + \frac{\beta}{2} (t_1 + t_2) \right] [t_1 - t_2] \end{aligned}$$

∴
$$Q = \frac{k_0 \cdot 2\pi L \left[1 + \frac{\beta}{2} (t_1 + t_2) \right] [t_1 - t_2]}{\ln(r_2 / r_1)} \quad \dots(2.62)$$

Integrating between r_1 and r , we obtain

$$Q = \frac{k_0 \cdot 2\pi L \left[1 + \frac{\beta}{2} (t_1 + t) \right] [t_1 - t]}{\ln(r / r_1)} \quad \dots(2.63)$$

Equating eqns. (2.62) and (2.63), we get

$$\frac{k_0 \cdot 2\pi L \left[1 + \frac{\beta}{2} (t_1 + t_2) \right] [t_1 - t_2]}{\ln(r_2 / r_1)} = \frac{k_0 \cdot 2\pi L \left[1 + \frac{\beta}{2} (t_1 + t) \right] [t_1 - t]}{\ln(r / r_1)}$$